

Fluid Kinematics

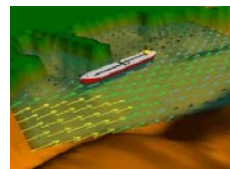
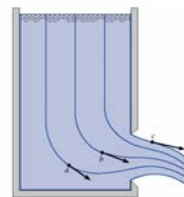


Fluid Flow Concepts and Reynolds Transport Theorem

- Descriptions of:
 - fluid motion
 - fluid flows
 - temporal and spatial classifications
- Analysis Approaches
 - Lagrangian vs. Eulerian
- Moving from a system to a control volume
 - Reynolds Transport Theorem

Fluid Motion

- Velocity field $\mathbf{V} = \frac{dx}{dt}\mathbf{i} + \frac{dy}{dt}\mathbf{j} + \frac{dz}{dt}\mathbf{k} \quad \mathbf{V} = u\mathbf{i} + v\mathbf{j} + w\mathbf{k}$
- Two ways to describe fluid motion
 - Lagrangian
 - Follow particles around
 - Eulerian
 - Watch fluid pass by a point or an entire region
 - Flow pattern
 - Streamlines – velocity is tangent to them



Analysis Approaches

- Lagrangian (system approach)
 - Describes a defined mass (position, velocity, acceleration, pressure, temperature, etc.) as functions of time
 - Track the location of a migrating bird`
- Eulerian
 - Describes the flow field (velocity, acceleration, pressure, temperature, etc.) as functions of position and time
 - Count the birds passing a particular location

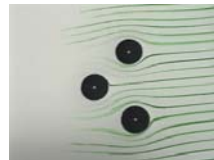
If you were going to study water flowing in a pipeline, which approach would you use? Eulerian

Descriptions of Fluid Motion

- streamline
 - has the direction of the velocity vector at each point
 - no flow across the streamline
 - steady flow streamlines are fixed in space
 - unsteady flow streamlines move
- pathline
 - path of a particle
 - same as streamline for steady flow
- streakline
 - tracer injected continuously into a flow
 - same as pathline and streamline for steady flow

Defined instantaneously

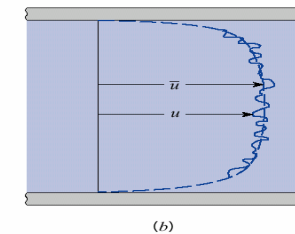
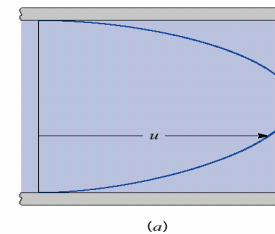
Defined as particle moves (over time)



Descriptors of Fluid Flows

Laminar vs Turbulent Flow

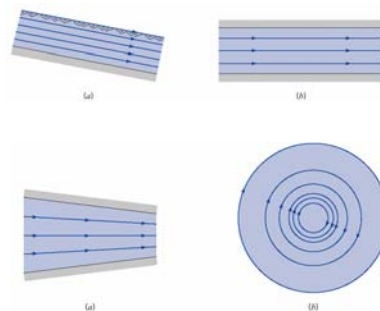
- Laminar flow
 - fluid moves along smooth paths
 - viscosity damps any tendency to swirl or mix
- Turbulent flow
 - fluid moves in very irregular paths
 - efficient mixing
 - velocity at a point fluctuates



Flow Patterns

Temporal/Spatial Classifications

- Uniform flow $\frac{\partial \mathbf{V}}{\partial s} = 0$
- Non-uniform flow $\frac{\partial \mathbf{V}}{\partial s} \neq 0$
- Steady flow $\frac{\partial \mathbf{V}}{\partial t} = 0$
- Unsteady flow $\frac{\partial \mathbf{V}}{\partial t} \neq 0$



Can turbulent flow be steady?
If averaged over a suitable time

Acceleration

- Cartesian coordinates $\mathbf{V} = u\mathbf{i} + v\mathbf{j} + w\mathbf{k}$ $\mathbf{a} = a_x\mathbf{i} + a_y\mathbf{j} + a_z\mathbf{k}$

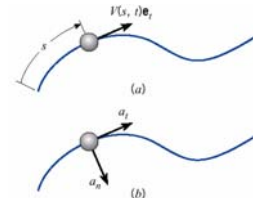
$$a_x = \frac{du}{dt} = \frac{\partial u}{\partial x} \frac{dx}{dt} + \frac{\partial u}{\partial y} \frac{dy}{dt} + \frac{\partial u}{\partial z} \frac{dz}{dt} + \frac{\partial u}{\partial t} = \underbrace{\frac{\partial u}{\partial x} u + \frac{\partial u}{\partial y} v + \frac{\partial u}{\partial z} w}_{\text{Convective}} + \underbrace{\frac{\partial u}{\partial t}}_{\text{Local}}$$

$$a_y = \frac{dv}{dt} = \frac{\partial v}{\partial x} \frac{dx}{dt} + \frac{\partial v}{\partial y} \frac{dy}{dt} + \frac{\partial v}{\partial z} \frac{dz}{dt} + \frac{\partial v}{\partial t} = \underbrace{\frac{\partial v}{\partial x} u + \frac{\partial v}{\partial y} v + \frac{\partial v}{\partial z} w}_{\text{Convective}} + \underbrace{\frac{\partial v}{\partial t}}_{\text{Local}}$$

$$a_z = \frac{dw}{dt} = \frac{\partial w}{\partial x} \frac{dx}{dt} + \frac{\partial w}{\partial y} \frac{dy}{dt} + \frac{\partial w}{\partial z} \frac{dz}{dt} + \frac{\partial w}{\partial t} = \underbrace{\frac{\partial w}{\partial x} u + \frac{\partial w}{\partial y} v + \frac{\partial w}{\partial z} w}_{\text{Convective}} + \underbrace{\frac{\partial w}{\partial t}}_{\text{Local}}$$

Acceleration

- Acceleration = rate of change of velocity
- Components:
 - Normal – changing direction
 - Tangential – changing speed



$$\mathbf{V} = V(s, t) \mathbf{e}_t$$

$$\frac{d\mathbf{V}}{dt} = \frac{dV}{dt} \mathbf{e}_t + V \frac{d\mathbf{e}_t}{dt}$$

$$\frac{dV}{dt} = V \frac{\partial V}{\partial s} + \frac{\partial V}{\partial t}$$

$$\frac{d\mathbf{e}_t}{dt} = \frac{V}{r} \mathbf{e}_n$$

$$\mathbf{a} = \left(V \frac{\partial V}{\partial s} + \frac{\partial V}{\partial t} \right) \mathbf{e}_t + \frac{V^2}{r} \mathbf{e}_n$$

The Dilemma

- The laws of physics in their simplest forms describe systems (the Lagrangian approach)
 - Conservation of Mass, Momentum, Energy
- It is impossible to keep track of the system in many fluids problems
- The laws of physics must still hold in a Eulerian world!
- We need some tools to bridge the gap

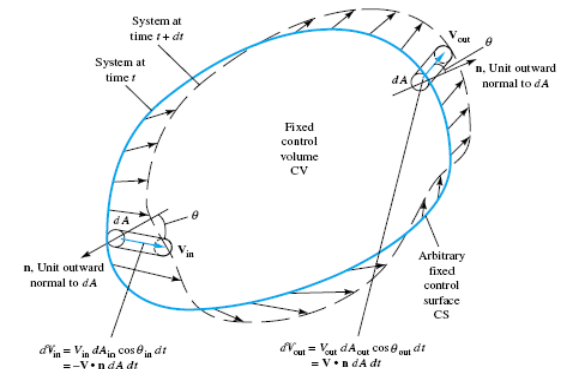
Reynolds Transport Theorem

- A moving system flows through the fixed control volume.
- The moving system transports extensive properties across the control volume surfaces.
- We need a bookkeeping method to keep track of the properties that are being transported into and out of the control volume

Control Volume Conservation Equation

B = Total amount of some properties in the system

b = Amount of the property per unit mass



$$\frac{d}{dt} (B_{\text{syst}}) = \frac{d}{dt} \left(\int_{\text{CV}} \beta \rho dV \right) + \int_{\text{CV}} \beta \rho (\mathbf{V} \cdot \mathbf{n}) dA$$

Rate of increase of the property in the system

= Rate of increase of the property in the control volume

+ Rate of efflux of the property across the control volume boundary

Application of Reynolds Transport Theorem

- Conservation of mass (for all species)
- Newton's 2nd law of motion (momentum)

$\mathbf{F} = m\mathbf{a}$

- First law of thermodynamics (energy)

continuity, momentum, and energy equations

Summary

- Reynolds Transport Theorem can be applied to a control volume of finite size
 - We don't need to know the flow details within the control volume!
 - We do need to know what is happening at the control surfaces.
- We will use Reynolds Transport Theorem to solve many practical fluids problems

Exercise:

4.34, 4.43, 4.49, 4.67

Flow Rate

- Volume rate of flow
 - Constant velocity over cross-section

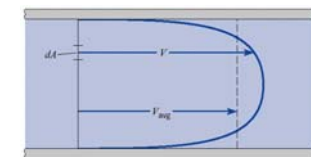
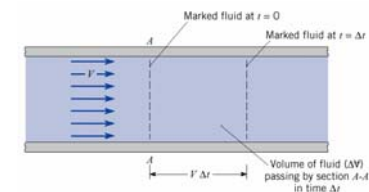
$$Q = VA$$

- Variable velocity

$$Q = \int_A V dA$$

- Mass flow rate

$$\dot{m} = \int_A \rho V dA = \rho \int_A V dA = \rho Q$$



Flow Rate

- Only x -direction component of velocity (u) contributes to flow through cross-section

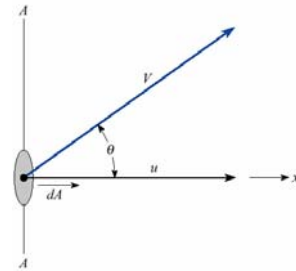
$$Q = \int_A V dA = \int_A u dA = \int_A V \cos \theta dA$$

or

$$Q = \int_A \mathbf{V} \cdot d\mathbf{A}$$

or

$$Q = \mathbf{V} \cdot \mathbf{A}$$

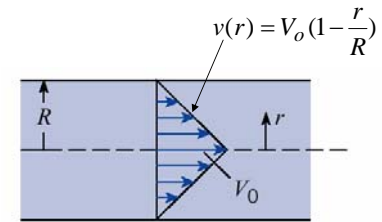


Example

- Find: \bar{V}

$$\begin{aligned} Q &= \int_A V dA = \int_0^R V_o (1 - r/R) 2\pi r dr \\ &= 2\pi V_o \left(\frac{r^2}{2} - \frac{r^3}{3R} \right) \Big|_0^R = 2\pi V_o \left(\frac{R^2}{2} - \frac{R^2}{3} \right) \\ &= \frac{1}{3} \pi V_o R^2 \end{aligned}$$

$$\frac{\bar{V}}{V_o} = \frac{Q}{A} V_o = \frac{\frac{1}{3} \pi V_o R^2}{\pi R^2 V_o} = \frac{1}{3}$$



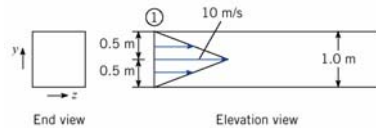
Example

- Find: Q, \bar{V}, \dot{m}

$$\begin{aligned} Q &= 2 \int_0^{0.5} V dy = 2 \int_0^{0.5} 20y dy \\ &= 40 \frac{y^2}{2} \Big|_0^{0.5} = 5 \text{ m}^3/\text{s} \end{aligned}$$

$$\bar{V} = \frac{Q}{A} = \frac{5}{1} = 5 \text{ m/s}$$

$$\dot{m} = \rho Q = 1.2 \times 5 = 6 \text{ kg/s}$$



Example

Given: $\mathbf{V} = 3t\mathbf{i} + xz\mathbf{j} + ty^2\mathbf{k}$

Find: Acceleration, \mathbf{a}

$$u = 3t; \quad v = xz; \quad w = ty^2$$

$$a_x = \frac{\partial u}{\partial x} u + \frac{\partial u}{\partial y} v + \frac{\partial u}{\partial z} w + \frac{\partial u}{\partial t} = 0(3t) + 0(xz) + 0(ty^2) + 3 = 3$$

$$a_y = \frac{\partial v}{\partial x} u + \frac{\partial v}{\partial y} v + \frac{\partial v}{\partial z} w + \frac{\partial v}{\partial t} = z(3t) + 0(xz) + x(ty^2) + 0 = 3zt + xy^2t$$

$$a_z = \frac{\partial w}{\partial x} u + \frac{\partial w}{\partial y} v + \frac{\partial w}{\partial z} w + \frac{\partial w}{\partial t} = 0(3t) + 2ty(xz) + 0(ty^2) + y^2 = 2xyzt + y^2$$

$$\mathbf{a} = a_x\mathbf{i} + a_y\mathbf{j} + a_z\mathbf{k} = 3t\mathbf{i} + (3zt + txy^2)\mathbf{j} + (2xyzt + y^2)\mathbf{k}$$